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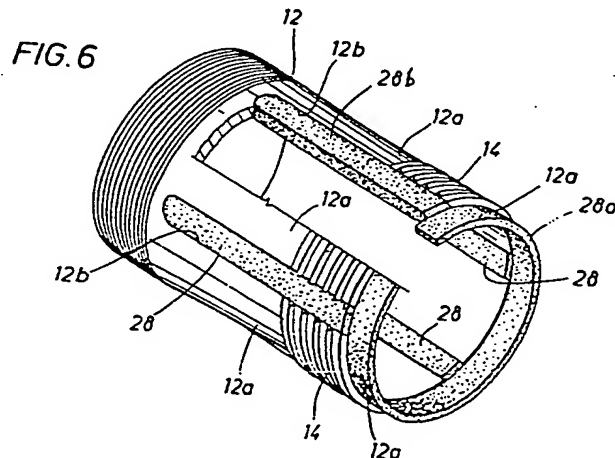
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(54) A latch, e.g. for latching tubing to a packer in a wellbore

(57) The latch 12 is adapted to be connected to a tubing string and will snap into a packer disposed in a wellbore. The latch 12 has an annular base with a plurality of axially extending fingers 12a separated, respectively, by a plurality of gaps, that snap into the packer, each finger having external threads 14 disposed around its periphery, each thread including an engaging surface and a disengaging surface, an angle between the surface of each disengaging thread relative to a direction being in a range from 20 to 28 degrees. An elastomeric barrier 28b occupies each gap between adjacent ones of the fingers 12a. A distance exists between each of the fingers 12a and an adjacent snap latch body, the fingers being adapted to bend and disengage across the distance, the distance being 2-1/2 to 3 times a particular distance that is required for the fingers to snap out of the packer in the wellbore when an operator pulls upwardly on the tubing. The geometry of the finger 12a is selected to minimize a lever arm extending between an axially applied force applied axially to the external threads 14 and a center of gravity of the finger at the base, the lever arm being approximately equal to zero. An outer diameter of each finger of the latch is greater than an inner diameter of an inner mandrel of the packer plus a tolerance stackup. The number of external threads 14 on the periphery of each finger 12a is minimized in order to further minimize the number of disengaging loads created when the latch is pulled out of the packer by the operator at the wellbore surface. This will assist in establishing as a certainty the pulling force which is required in order to pull the tubing and the snap latch out of the packer in the wellbore and the pushing force required to snap the tubing and the snap latch into the packer in the wellbore.



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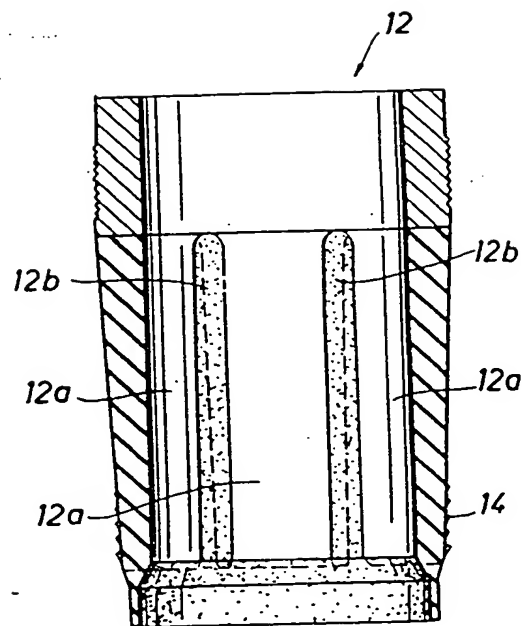
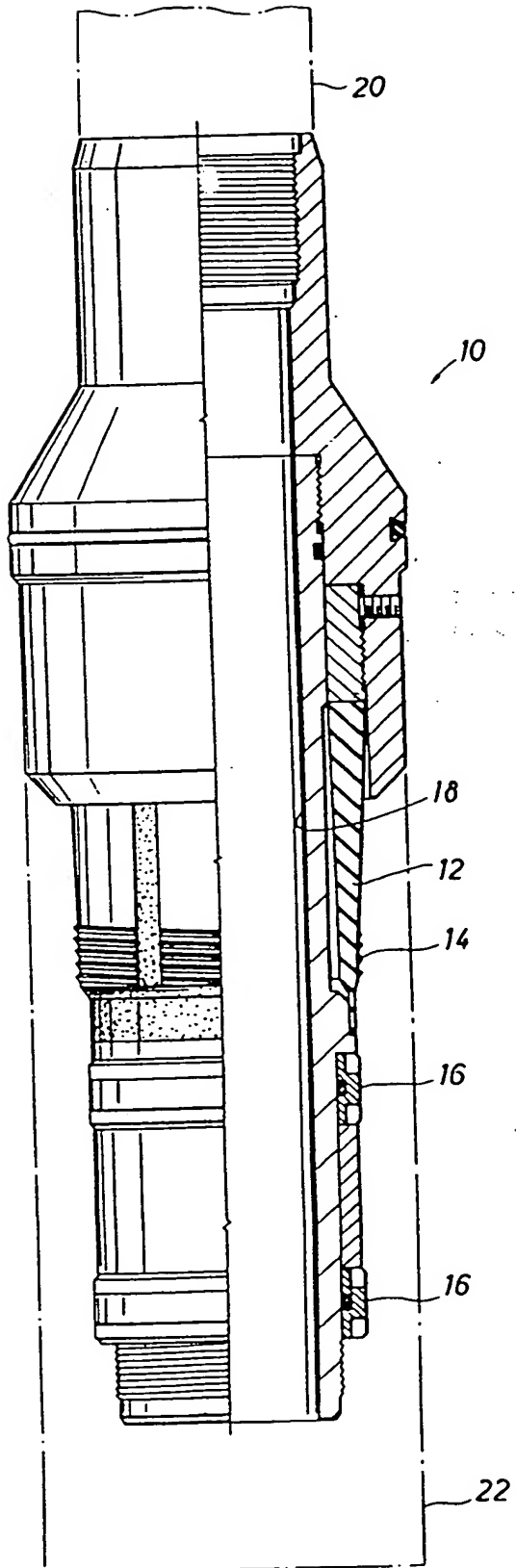


FIG. 2

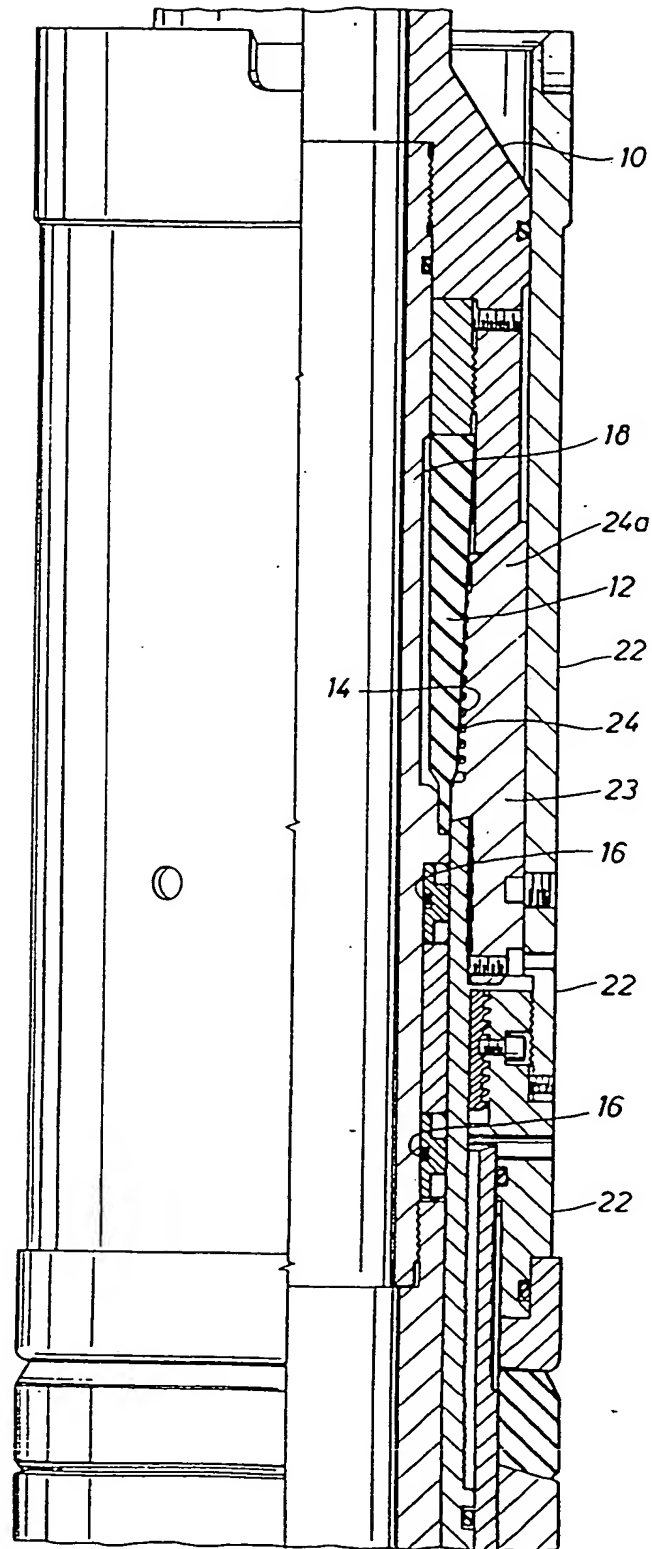


FIG. 4

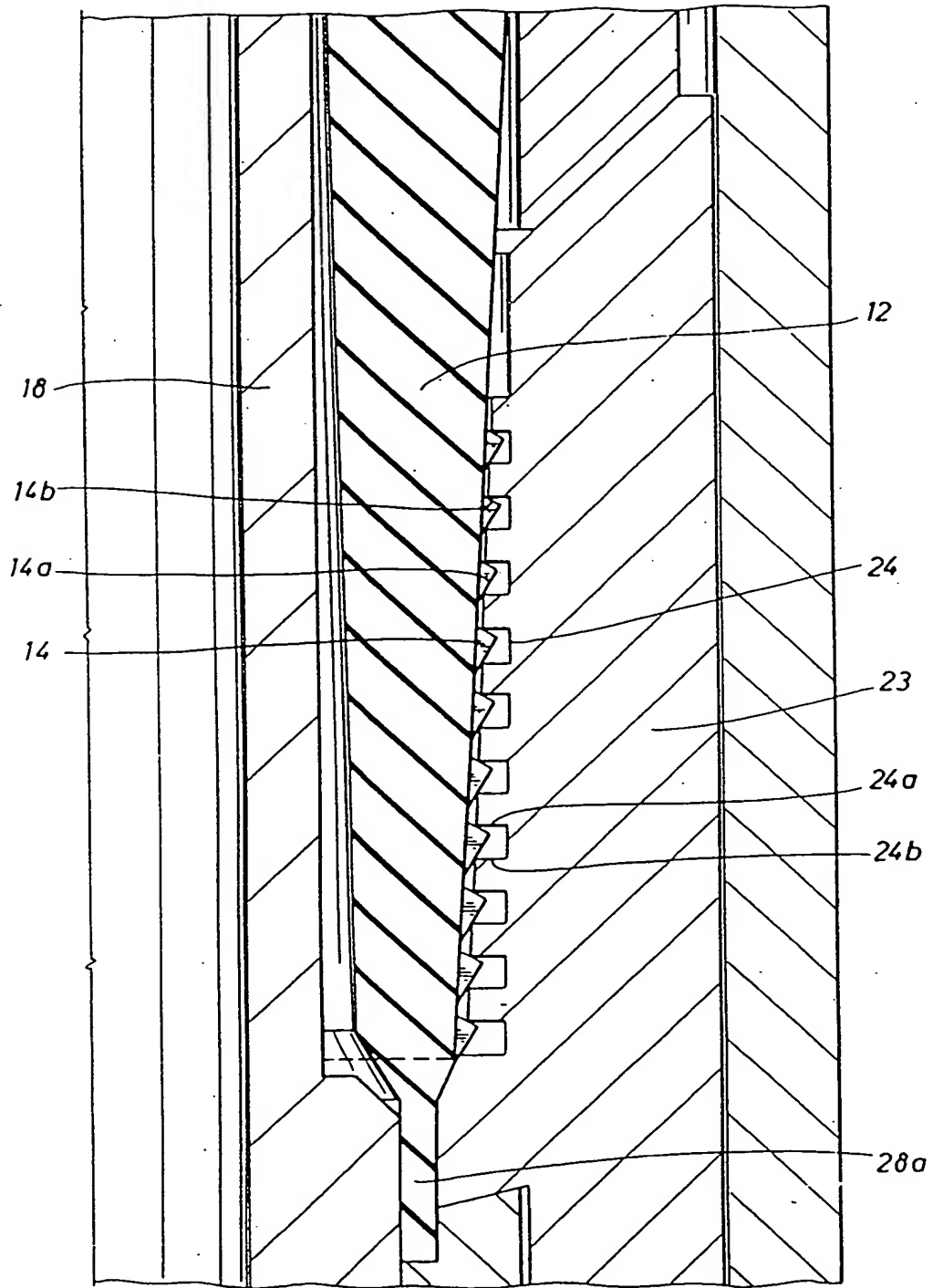


FIG. 5
(PRIOR ART)

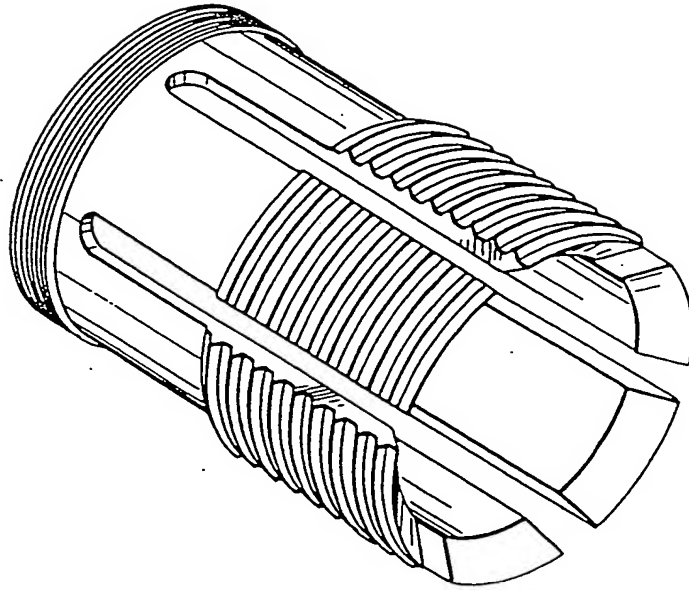
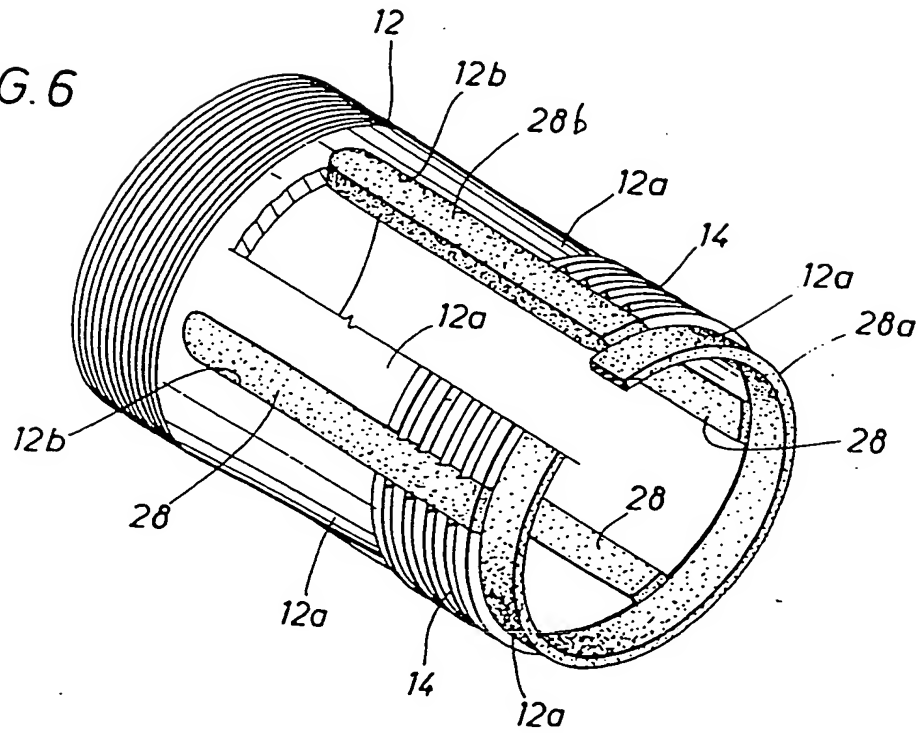


FIG. 6



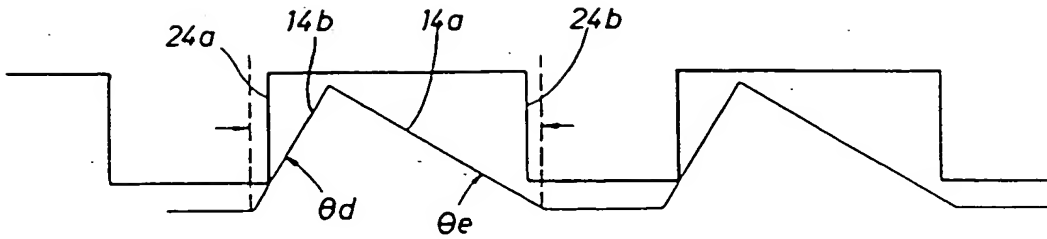


FIG. 7

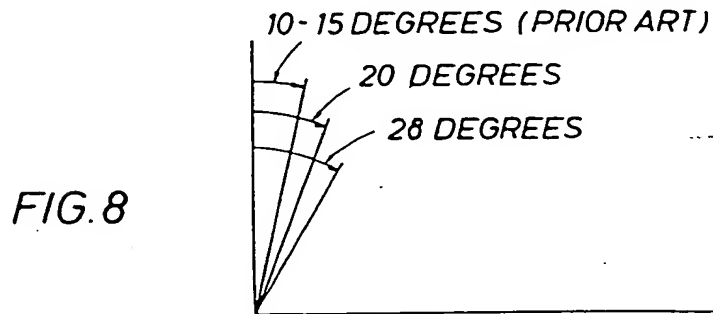


FIG. 8

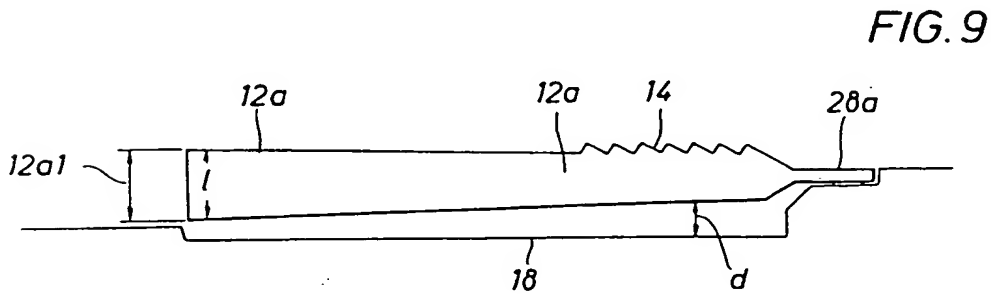


FIG. 9

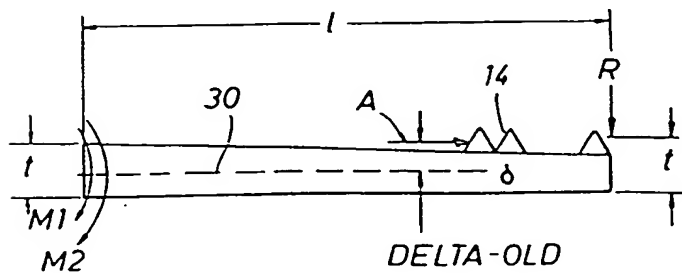


FIG. 10
(PRIOR ART)

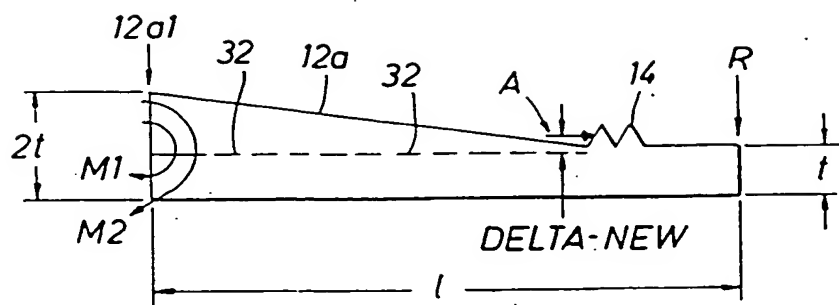


FIG.11

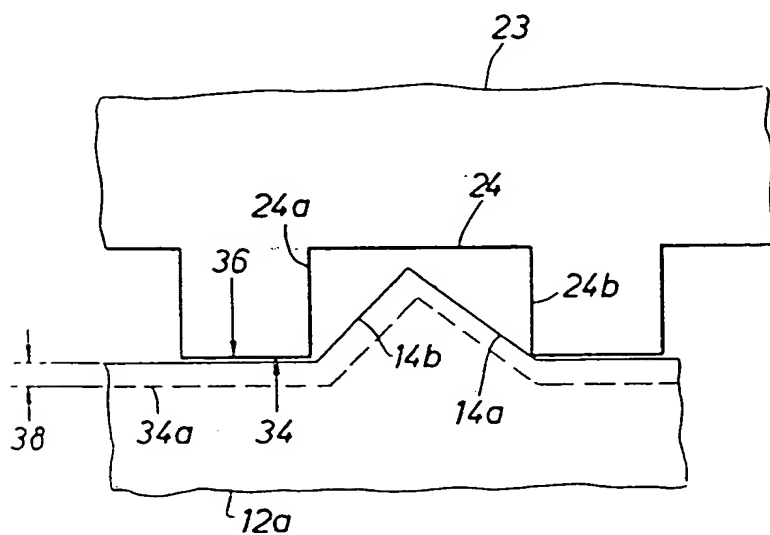


FIG. 12

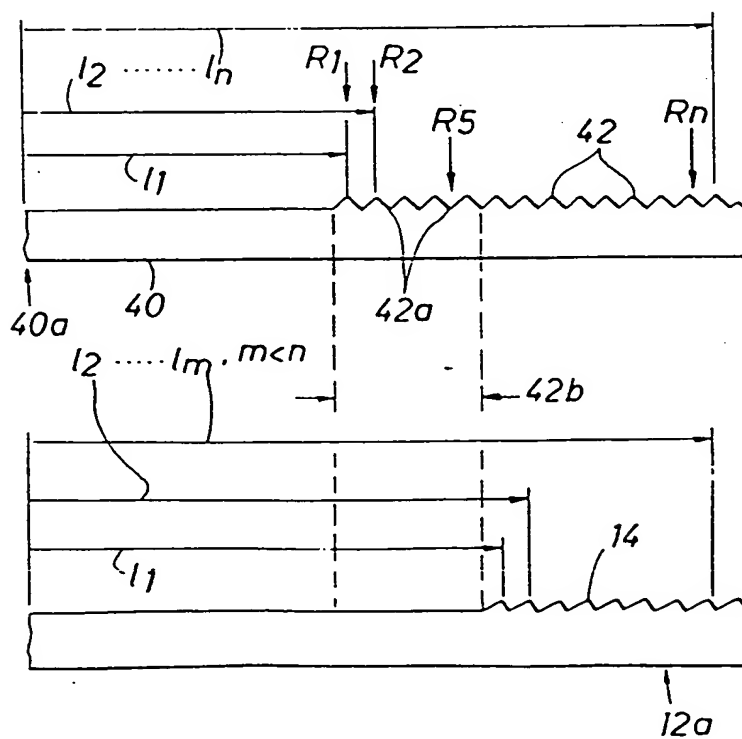


FIG. 13
(PRIOR ART)

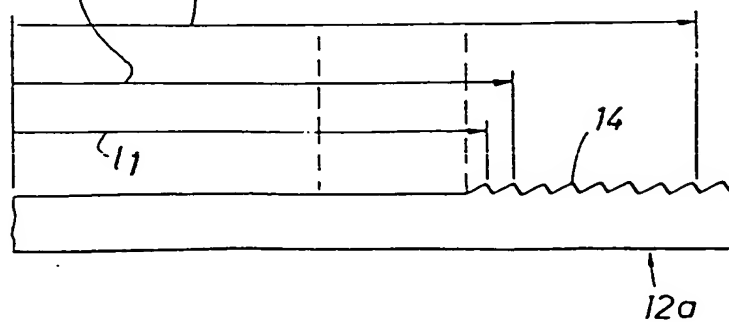


FIG. 14

A LATCH, E.G. FOR LATCHING TUBING TO A PACKER IN A WELLBORE

The present invention relates to a latch adapted for interconnecting a first part to a second part in a wellbore: in particular, the invention provides a latch adapted to be connected on one end to a tubing string for latching the tubing string to a packer disposed in a wellbore and for simultaneously locating a plurality of seals inside the packer, the seals providing a fluid tight seal between an internal part of the packer and an external part of the tubing.

A snap latch is used to interconnect a tubing string to a packer disposed in a wellbore. An operator at the surface of the wellbore connects the snap latch to a tubing string and lowers the snap latch and accompanying tubing string into the wellbore until the snap latch locates the packer already disposed in the wellbore. The snap latch includes a plurality of flanges, and the flanges are designed to snap into an internal part of the packer. When the flanges of the snap latch are snapped into the packer, wellbore operations may commence. It may be necessary to disconnect the snap latch from the packer and pull the tubing string out of the wellbore to a surface of the wellbore. In this case, the operator at the surface pulls upwardly on the tubing. In response to the pull upwardly on the tubing, since the snap latch is connected to the tubing, the flanges of the snap latch are supposed to bend inwardly and disconnect from the internal part of the packer. The flanges include a plurality of externally disposed threads (hereinafter called external threads) which are disposed on an external periphery of the flange, the external threads on the flange mating with a corresponding plurality of internally disposed threads (hereinafter called internal threads) disposed around an internal periphery of the internal part of the packer mandrel. The external threads of the flange are shaped differently than the internal threads of the packer. The external threads include a disengaging surface disposed at a disengaging angle relative to a vertical and an engaging surface disposed at an engaging angle relative to a vertical, whereas the internal threads are primarily square-shaped, each square shaped internal thread including a rising surface and a falling surface. When the snap latch is snapped into the packer, the disengaging surface of an external thread is in contact with the rising surface of an internal thread, and the engaging surface of the external thread is in contact with the falling surface of the internal thread.

However, a first problem is encountered when wellbore debris is disposed between the disengaging surface of the external threads of a flange and the rising surface of the internal threads of the packer. The debris will actually increase the frictional contact between the disengaging surface and the rising surface. This increased frictional contact

5 can cause the flange to bend outwardly instead of inwardly when the operator pulls upwardly on the tubing at the wellbore surface, and this outward bending of the flange makes it more difficult for the operator to disconnect the snap latch from the packer. Therefore, it becomes more difficult to disconnect the tubing string from the packer and pull the tubing string out of the wellbore. Furthermore, if the tubing string cannot be
10 disconnected from the packer during the pull upwardly on the tubing in the wellbore, a safety shear sub, connected in the tool string below the snap latch, may snap.

In addition, a second problem is encountered when wellbore debris occupies the gaps disposed between the flanges of the snap latch and accumulates on an undersurface of the flange. Debris in this location also makes it more difficult for the operator to pull
15 upwardly on the tubing and disconnect the snap latch from the packer. In addition, when further debris accumulates in a space disposed between the flange of the latch and a snap latch body, this space is oftentimes too small and, as a result, inward bending of the flange is inhibited due to the accumulation of the further debris in the space.

In addition, a third problem is encountered due to a lengthy radial distance or lever arm
20 between the center of gravity of the flange and an axially applied force acting on the threads on the flange, this lengthy lever arm creating an undesired auxiliary bending moment in addition to the normal and expected bending moment caused by a transversely applied force acting on the engaging surface of the threads of the flange. Recalling that the objective of the snap latch of the present invention is to make it harder to disengage
25 the tubing from the packer and easier to engage the tubing to the packer, the undesired auxiliary bending moment makes it easier to disengage the tubing from the packer and harder to engage the tubing to the packer, which is the opposite of what is required.

In addition, a fourth problem is encountered due to the tolerance or error caused by machining the outer diameter of the latch flange. When machining the flange, the outer
30 diameter (OD) of the flange, where the engaging and disengaging surfaces of the external threads of the flange of the latch are located, is not exactly equal to the inner diameter (ID) of a mandrel, where the rising and falling surfaces of the square shaped internal threads are located. The difference between the OD of the flange and the ID of the mandrel is the tolerance or error caused during machining of the flange. The amount of
35 engagement between the OD and the ID takes on a range of values; sometimes the OD and the ID are more than touching, and sometimes there is a gap between the OD and the ID. Consequently, the engaging and disengaging surfaces of the external threads of the flange do not exactly mate or come into complete contact with the rising and falling surfaces of the internal threads of the mandrel.

5 In addition, a fifth problem is encountered due to the excessive number of external
threads located the exterior surface of the flanges of the snap latch. Each external thread
creates a normal (and undesired) bending moment at the base of each flange of the snap
latch. Generally speaking, the greater the number of teeth of the external threads that
10 exist on the exterior surface of the flanges, the larger the range of bending moments that
exist and the larger the range of engaging and disengaging loads that exist. If there are
too many external threads disposed on the exterior surface of the flanges of the latch,
there will be an excessive number of bending moments and associated engaging and
disengaging loads, and this excessive number of bending moments and associated
15 engaging and disengaging loads will complicate the process of disconnecting the snap
latch and associated tubing from the packer disposed in the wellbore. It is therefore more
difficult for the operator to disconnect the tubing from the packer and pull the tubing to a
surface of the wellbore.

Accordingly, a new snap latch design is needed to alleviate at least some of the above
referenced problems so that the operator at the wellbore surface will know for certain that
20 a predetermined first number of pounds is required to snap the latch into the packer and
that a predetermined second number of pounds (greater than the first number) is required
to snap the latch out of the packer disposed in the wellbore.

Accordingly, it is a primary object of the present invention to provide such a latch.

According to a first aspect of the present invention, there is provided a latch, which is
25 adapted to be connected to a tubing string, that will snap into a packer disposed in a
wellbore, a predetermined first number of pounds being required to snap the latch into
the packer and a predetermined second number of pounds, greater than the first number,
being required to pull the latch out of the packer in the wellbore, the latch having a
plurality of flanges that snap into the packer, each flange having external threads
30 disposed around its periphery, each thread including an engaging surface and a
disengaging surface, an angle between the surface of each disengaging thread relative to a
vertical, for a specific size snap latch, being a specific value, the specific value being
within a range from 20 degrees to 28 degrees.

A further aspect of the present invention provides a latch, which is adapted to be
35 connected to a tubing string, that will snap into a packer disposed in a wellbore, a
predetermined first number of pounds being required to snap the latch into the packer and
a predetermined second number of pounds, greater than the first number, being required
to pull the latch out of the packer in the wellbore, the latch having a plurality of flanges
separated, respectively, by a plurality of gaps, that snap into the packer, each flange
40 having external threads disposed around its periphery, each thread including an engaging

5 surface and a disengaging surface, an angle between the surface of each disengaging thread relative to a vertical, for a specific size snap latch, being a specific value, the specific value being in a range from 20 to 28 degrees, an elastomeric barrier occupying the gap between adjacent ones of the flanges.

10 A still further aspect of the present invention provides a latch, which is adapted to be connected to a tubing string, that will snap into a packer disposed in a wellbore, a predetermined first number of pounds being required to snap the latch into the packer and a predetermined second number of pounds, greater than the first number, being required to pull the latch out of the packer in the wellbore, the latch having a plurality of flanges separated, respectively, by a plurality of gaps, that snap into the packer, each flange
15 having external threads disposed around its periphery, each thread including an engaging surface and a disengaging surface, an angle between the surface of each disengaging thread relative to a vertical, for a specific size snap latch, being a specific value, the specific value being in a range from 20 to 28 degrees, an elastomeric barrier occupying the gap between adjacent ones of the flanges, a distance existing between each of the
20 flanges and an adjacent snap latch body, the flanges being adapted to bend and disengage across the distance, the distance being 2-1/2 to 3 times a particular distance that is required for the flange to snap out of the packer in the wellbore when an operator pulls upwardly on the tubing.

Yet another aspect of the present invention provides a latch, which is adapted to be
25 connected to a tubing string, that will snap into a packer disposed in a wellbore, a predetermined first number of pounds being required to snap the latch into the packer and a predetermined second number of pounds, greater than the first number, being required to pull the latch out of the packer in the wellbore, the latch having a plurality of flanges separated, respectively, by a plurality of gaps, that snap into the packer, each flange
30 having external threads disposed around its periphery, each thread including an engaging surface and a disengaging surface, an angle between the surface of each disengaging thread relative to a vertical, for a specific size snap latch, being a specific value, the specific value being in a range from 20 to 28 degrees, an elastomeric barrier occupying the gap between adjacent ones of the flanges, a distance existing between each of the
35 flanges and an adjacent snap latch body, the flanges being adapted to bend and disengage across the distance, the distance being 2-1/2 to 3 times a particular distance that is required for the flange to snap out of the packer in the wellbore when an operator pulls upwardly on the tubing, each flange having a base which has a particular thickness, the particular thickness being selected to minimize a lever arm extending between an axially

- 5 applied force applied axially to the external threads and a center of gravity of the flange at the base.

Another aspect of the present invention provides a latch, which is adapted to be connected to a tubing string, that will snap into a packer disposed in a wellbore, a predetermined first number of pounds being required to snap the latch into the packer and
10 a predetermined second number of pounds, greater than the first number, being required to pull the latch out of the packer in the wellbore, the latch having a plurality of flanges separated, respectively, by a plurality of gaps, that snap into the packer, each flange having external threads disposed around its periphery, each thread including an engaging surface and a disengaging surface, an angle between the surface of each disengaging
15 thread relative to a vertical, for a specific size snap latch, being a specific value, the specific value being in a range from 20 to 28 degrees, an elastomeric barrier occupying the gap between adjacent ones of the flanges, a distance existing between each of the flanges and an adjacent snap latch body, the flanges being adapted to bend and disengage across the distance, the distance being 2-1/2 to 3 times a particular distance that is
20 required for the flange to snap out of the packer in the wellbore when an operator pulls upwardly on the tubing, each flange having a base which has a particular thickness, the particular thickness being selected to minimize a lever arm extending between an axially applied force applied axially to the external threads and a center of gravity of the flange at the base, an outer diameter of each flange of the latch being greater than an inner diameter
25 of an inner mandrel of the packer plus a tolerance stackup, and the number of external threads on the periphery of each flange being minimized in order to further minimize the number of bending moments created when the latch is pulled out of the packer by the operator at the wellbore surface.

A preferred embodiment of the latch of the present invention has the following special
30 characteristics:

1. The latch has a plurality of flanges separated, respectively, by a plurality of gaps, that snap into the packer, each flange having external threads disposed around its periphery, each thread including an engaging surface and a disengaging surface, an angle between
35 each disengaging surface and a vertical, for a specific size snap latch, being a specific value, the specific value being within a range between 20 degrees and 28 degrees,
2. An elastomeric barrier occupies the gap between adjacent ones of the flanges,
3. A distance exists between each of the flanges and an adjacent snap latch body, the flanges being adapted to bend and disengage across the distance, the distance being 2-1/2

5 to 3 times a particular distance that is required for the flange to snap out of the packer in the wellbore when an operator pulls upwardly on the tubing,

4. Each flange has a geometry that minimizes an undesirable auxiliary bending moment. As a result, the flange has a base which has a particular thickness, the particular thickness being selected to minimize a lever arm extending between an axially applied force applied
10 axially to the external threads and a center of gravity of the flange at the base such that the lever arm is approximately equal to zero,

5. An outer diameter of each flange of the latch is greater than an inner diameter of an inner mandrel of the packer plus a tolerance stackup, and

6. The number of external threads on the periphery of each flange of the snap latch of the
15 present invention is reduced, relative to the number of external threads on the periphery of each flange of the prior art snap latch, in order to achieve a smaller range of bending moments and therefore a smaller range of engaging and disengaging loads when the latch is pulled out of the packer by the operator at the wellbore surface. This will assist in establishing the second number of pounds of pulling force which is required in order to
20 pull the tubing and the snap latch of the present invention out of the packer in the wellbore.

Further scope of applicability of the present invention will become apparent from the detailed description presented hereinafter. It should be understood, however, that the detailed description and the specific examples, while representing a preferred
25 embodiment of the present invention, are given by way of illustration only, since various changes and modifications within the spirit and scope of the invention will become obvious to one skilled in the art from a reading of the following detailed description.

A full understanding of the present invention will be obtained from the detailed description of the preferred embodiment presented hereinbelow, and the accompanying
30 drawings, which are given by way of illustration only and are not intended to be limitative of the present invention, and wherein:

figure 1 illustrates a construction of the snap latch seal locator of the present invention;

figure 2 illustrates the snap latch seal locator of figure 1 inserted and snapped into a packer disposed in a wellbore;

35 figure 3 illustrates a construction of the latch portion of the snap latch seal locator of figure 1;

figure 4 illustrates an expanded view of the external threads on the latch portion of the snap latch mating with the internal threads on the packer;

5 figure 5 illustrates a three dimensional drawing of a prior art latch portion of a prior art snap latch;

figure 6 illustrates a three dimensional drawing of the latch portion of figure 3 of the snap latch seal locator of figure 1 of the present invention, figure 6 illustrating an elastomeric barrier inserted within each of the gaps disposed between adjacent flanges of the latch
10 portion of the snap latch seal locator;

figures 7 illustrates the external threads on a flange of the latch portion of figure 6 in contact with the internal threads on the mandrel of the packer in the wellbore;

figure 8 illustrates the number of degrees of a disengaging angle which extends between the disengaging surface of one of the external threads of the flange of the latch portion of
15 figure 6 and a vertical line;

figure 9 illustrates a space or distance "d" which exists between one of the flanges of the latch portion of the snap latch of figure 6 and a snap latch body of the snap latch, this distance "d" being 2-1/2 to 3 times the distance which is required for the flange of the latch portion to bend and disengage with the square threads of the mandrel of the packer;

20 figure 10 illustrates a flange associated with a snap latch of the prior art, there being an undesired auxiliary moment created by an axially applied force "A" and a lever arm "delta-old";

figure 11 illustrates a flange associated with the latch portion of the snap latch seal locator of the present invention of figure 1, the undesired auxiliary moment of figure 10 being
25 approximately equal to zero because a new lever arm "delta-new" is approximately equal to zero;

figure 12 illustrates an external thread on a flange of a latch portion of the snap latch of figure 1 in contact with an internal thread on an internal periphery of a mandrel of the packer, this figure illustrating the following basic principle in accordance with one aspect
30 of the present invention: the OD of the latch flange > ID of the mandrel on the packer + tolerance stackup;

figure 13 illustrates the external threads on a flange of a prior art latch portion of a prior art snap latch seal locator; and

figure 14 illustrates the external threads on a flange of the latch portion of the snap latch
35 of the present invention.

Referring to figure 1, a snap latch seal locator 10 in accordance with the present invention is illustrated. In figure 1, the snap latch seal locator 10 of the present invention is adapted to be interconnected between a tubing string 20 and a packer assembly 22. When the

5 tubing string 20 is connected to the snap latch seal locator 10, the snap latch seal locator is adapted to be lowered into a borehole and the seal locator 10 is "snapped" in place into the packer assembly 22. When desired, a pull upwardly on the tubing string 20 by an operator at the wellbore surface should "unsnap" the snap latch seal locator 10 from the packer assembly 22. At this point, the tubing 20 and snap latch seal locator 10 can be
10 pulled uphole to a surface of the wellbore.

In figure 1, the snap latch seal locator 10 includes the latch portion 12 of the snap latch seal locator 10, the latch portion 12 being hereinafter called "a snap latch 12". The snap latch 12 includes a plurality of external threads 14 disposed around an external periphery of the snap latch 12. Actually, the snap latch 12 includes a plurality of flanges, and the
15 external threads 14 are disposed around the external periphery of the flanges of the snap latch 12. The snap latch seal locator 10 further includes a plurality of seals 16 which are adapted to be located within a packer assembly and seals the interior of the packer assembly and seal locator 10 from the exterior thereof. The snap latch 12 of figure 1 further includes a plurality of elastomeric barriers disposed, respectively, within a
20 corresponding plurality of gaps located between adjacent flanges of the snap latch; thus, the cross-sectioning of the snap latch 12 as shown in figure 1 actually refers to one of the elastomeric barriers disposed in one of the gaps. The elastomeric barrier, the flanges, and the gaps of the snap latch 12 can be seen in greater detail in figure 6 and will be discussed in greater detail with reference to figure 6. The snap latch 12 is spaced from a
25 snap latch body 18 by a distance "d", shown in greater detail in figure 10.

Referring to figure 2, the snap latch seal locator 10 of figure 1 is shown inserted into a packer assembly

In figure 2, the snap latch seal locator 10 of figure 1 has been inserted into the packer assembly 22, and the seals 16 of the seal locator 10 are shown in figure 2 to be disposed
30 in sealing engagement with an internal part of the packer assembly 22. Recalling that the snap latch 12 has a plurality of flanges and that the plurality of external threads 14 are disposed around an external periphery of the flanges, in figure 2, the external threads 14 on the flanges of the snap latch 12 are shown to be firmly engaged with a plurality of internal threads 24 disposed within the packer assembly 22. More particularly, the
35 packer assembly 22 includes a mandrel 23 and the mandrel 23 has the internal threads 24 disposed around its internal periphery. Note that the internal threads 24 of the mandrel 23 are each square shaped and have two surfaces, a rising surface and a falling surface; however, the external threads 14 of the snap latch 12 also have two surfaces, an engaging surface and a disengaging surface. This construction will be discussed later in
40 this specification.

5 Referring to figure 3, the snap latch 12 of figures 1 and 2 is illustrated again and notice
that the snap latch 12 includes the external threads 14. The snap latch 12 includes a
plurality of flanges 12a, each of the flanges 12a being separated from an adjacent flange
12a by a gap 12b. For reasons which will be discussed in more detail later in this
10 specification, an elastomeric barrier (shown in figure 3, but better illustrated in figure 6)
is disposed within each of the gaps 12b between adjacent flanges 12a of the snap latch
12.

Referring to figure 4, an expanded view of the snap latch 12 of the snap latch seal locator
10 of the present invention is illustrated. In figure 4, the expanded view of the snap latch
12 provides a better view of how the external threads 14 on the snap latch 12 mate with
15 the internal threads 24 of the mandrel 23. In figure 4, the external threads 14 disposed
around an external periphery of each flange 12a of snap latch 12 include an engaging
surface 14a and a disengaging surface 14b. However, the internal threads 24 disposed
around an internal periphery of the mandrel 23 include a rising surface 24a and a falling
surface 24b.

20 In order to pull the snap latch seal locator 10 out of the packer assembly 22 of figure 2,
the disengaging surface 14b of the external threads 14 on each flange 12a of the snap
latch 12 must disengage from the rising surface 24a of the internal threads 24 on the
mandrel 23, and, in order to "snap" the snap latch seal locator 10 into the packer
assembly 22 of figure 2, the engaging surface 14a of the external threads 14 on each
25 flange 12a of the snap latch 12 must engage with the falling surface 24b of the internal
threads 24 of the mandrel 23.

However, one major objective of the snap latch seal locator 10 of the present invention is
as follows: the amount of pounds of force required to "snap" the snap latch seal locator
10 into the packer assembly 22 of figure 2 and connect the tubing string 20 to the packer
30 assembly 22 is 3000 pounds; however, the amount of pounds of force that is required to
bend the flanges 12a of the snap latch 12, pull the snap latch seal locator 10 out of the
packer assembly 22 of figure 2, and pull the tubing string 20 out of the wellbore is 15000
pounds.

Therefore, in order to satisfy this major objective, the angle of the disengaging surface
35 14b of the external threads 14 on each flange 12a of the snap latch 12, relative to a
vertical, is very important, and the angle of the engaging surface 14a of the external
threads 14 relative to a vertical is also very important. This will be explained in more
detail later in this specification.

Figure 5 illustrates a three dimensional drawing of a prior art snap latch.

5 Figure 6 illustrates a three dimensional drawing of the snap latch 12 of the snap latch seal locator 10 of the present invention. In figure 6, recall that six (6) improvements were made to the snap latch seal locator of the prior art when designing the snap latch seal locator 10 of the present invention.

10 A first improvement to the snap latch seal locator of the prior art relates to an elastomeric barrier 28 disposed within each of the gaps 12b between each of the flanges 12a of the snap latch 12 of the snap latch seal locator 10 of the present invention. The elastomeric barrier 28 includes a top ring-like part 28a and a plurality of extension members 28b integrally connected to the top ring-like part 28a, the plurality of extension members 28b being adapted to be inserted within the plurality of gaps 12b disposed between the
15 plurality of flanges 12a of the snap latch 12. The elastomeric barrier 28 closes each of the gaps 12b between each of the flanges 12a of the snap latch 12 and prevents debris, originating from outside the snap latch 12, from entering an interior of the snap latch 12. Without this elastomeric barrier 28, this debris from outside the snap latch 12 will pass through the gaps 12b between each of the flanges 12a and will begin to build up on an
20 underside on the interior of each flange 12a. A debris layer will be deposited on the underside of each flange 12a, and, when the operator at the wellbore surface pulls upwardly on the tubing 20 and tries to disengage the tubing 20 from the packer assembly 22, this debris layer deposited on the underside of each flange 12a will tend to prevent each flange 12a from bending inwardly to a desired extent which is required to disengage
25 the disengaging surface 14b of the external threads 14 on the flange from the rising surface 24a of the internal threads 24 on the mandrel 23. As a result, without the elastomeric barrier 28, more than 15000 pounds of disengaging force will be required to pull the tubing 20 out of the packer assembly 22 of figure 1. Note that the elastomeric barrier 28 includes a tip end 28a which prevents the debris from entering an interior of
30 the snap latch 12 (see in particular the location of the tip end 28a of elastomeric barrier 28 in figures 1, 2, and 4 of the drawings and note how the tip end 28a prevents the passage of any debris from outside to inside the snap latch 12).

Referring to figures 7 and 8, an expanded view of one external thread 14 which is disposed on each flange 12a of the snap latch 12 of the snap latch seal locator 10 of the
35 present invention and one internal thread 24 on the mandrel 23 of the packer assembly 22 is illustrated.

In figure 7, a disengaging angle θ_d is defined to be the angle in degrees between each of the disengaging surfaces 14b of each of the external threads 14 on each flange 12a of the snap latch 12 and a vertical line, whereas the engaging angle θ_e is defined to be the
40 angle in degrees between each of the engaging surfaces 14a of each of the external

5 threads 14 on each flange 12a of the snap latch 12 and a vertical line, as shown in figure 7.

A second improvement to the snap latch seal locator 10 in accordance with the present invention relates to the disengaging angle θ_d .

10 In figure 8, in the prior art snap latch shown in figure 5, the disengaging angle θ_d was approximately ten (10) degrees. However, gravel and other debris would enter the space between each disengaging surface 14b of each external thread and the rising surface 24a of each internal thread 24. This debris would increase the coefficient of friction between surface 14b and surface 24a. As a result, the amount of pull out force required to unsnap the snap latch 12 from the packer assembly 22 was greater than 15000 pounds. Recall
15 that it is necessary to maintain the pull out force at a constant 15000 pounds.

However, in accordance with the second improvement of the present invention, for a specific size snap latch, the disengaging angle θ_d , between each of the disengaging surfaces 14b and a vertical line for each external thread 14 on each flange 12a of the snap latch 12 of the present invention shown in figure 6, is a specific value, the specific value
20 lying within a range between twenty (20) degrees and twenty-eight (28) degrees. Recalling that one major objective of the snap latch seal locator 10 is to require a 15000 pound pulling force to unsnap the snap latch 12 from the packer assembly 22 in figure 1, since the disengaging angle θ_d , for a specific size snap latch, is a specific value and that specific value now lies in a range between 20 and 28 degrees, the operator at the
25 wellbore surface can now be assured that an approximately constant 15000 pounds is required to unsnap the snap latch 12 from the packer assembly 22 and pull the tubing string 20 to the surface of the wellbore.

Referring to figure 9, a distance "d" exists between each flange 12a of the snap latch 12 and the snap latch body 18 (see figure 1), the distance "d" increasing from the base 12a1
30 of the flange 12a to the tip end 28a of the flange.

A third improvement to the snap latch seal locator 10 in accordance with the present invention relates to this distance "d" shown in figure 9. The third improvement is as follows: the distance "d" must be two and one half (2-1/2) to three (3) times a distance "x", where the distance "x" is defined to be the distance between the flange 12a and the
35 snap latch body 18 that the flange 12a of snap latch 12 bends inwardly toward snap latch body 18 when the 15000 pound upward pulling force is applied to the tubing 20 at the wellbore surface for the purpose of disengaging the disengaging surface 14b of the external threads 14 on flange 12a from the internal threads 24 of the mandrel 23 of the packer assembly 22.

5 As a result of this specific distance "d", if any excess debris accumulates within the space defined by distance "d" between flange 12a and snap latch body 18 in figure 9, that debris will not prevent or inhibit the flange 12a from bending inwardly the required distance "x" in response to the 15000 pound pulling force on tubing 20 since the distance "d" is 2-1/2 to 3 times the distance "x" and is therefore more than enough distance for
10 each of the flanges 12a to bend and disengage from the internal threads 24.

In addition, since the geometry of the snap latch of the present invention is changed to minimize the auxiliary bending moment, the thickness or length "l" of the base 12a1 of the flange 12a in figure 9 is thicker than the thickness of the flange at the tip end 28a of the flange 12a.

15 A fourth improvement to the snap latch seal locator 10 of the present invention relates to the thickness or length "l" of the base 12a1 of each flange 12a of the snap latch 12. Recalling that the disengaging angle θ_d is the angle between the engaging surface 14b of each external thread 14 and a vertical, the higher the disengaging angle θ_d is, in degrees, the longer the length "l", or thickness, of the base 12a1 must be in order to
20 maintain constant the 15000 pound pull out force, the 15000 pound pull out force being required to unsnap the snap latch 12 from the packer assembly 22 when the operator at the wellbore surface attempts to pull the tubing string 20 out of the wellbore. The reason for this constraint is simple: the higher the disengaging angle, the easier it would be to disengage the snap latch 12 from the packer assembly 22. As a result, if the thickness or
25 length "l" of the base 12a1 were not adjusted when the disengaging angle is increased, the pull out force would be quite a bit less than the desired 15000 pounds. Therefore, if the disengaging angle θ_d increases, the base 12a1 of each flange 12a must increase in length "l" or thickness in order to maintain constant the 15000 pound pull out force which is required to unsnap the snap latch 12 from the packer assembly 22 and pull the
30 tubing string 20 out of the wellbore.

Referring to figures 10 and 11, a flange associated with a snap latch of the prior art is illustrated in figure 10, there being an undesired auxiliary bending moment created by an axially applied force "A" and a lever arm "delta-old", and a flange associated with the snap latch of the snap latch seal locator of the present invention is illustrated in figure 11,
35 the undesired auxiliary bending moment of figure 10 being approximately equal to zero because a new lever arm "delta-new" is approximately equal to zero.

A fifth improvement to the snap latch seal locator 10 of the present invention relates to the elimination of the undesired auxiliary bending moment "M2" by decreasing the length of the lever arm "delta-old" to an amount which is approximately equal to zero.

5 In figure 10, when the operator at the wellbore surface applies the required 15000 pound pulling force upwardly to the tubing string 20, two forces are applied to each flange 12a of the snap latch 12 of the snap latch seal locator 10: (1) a first force "R", which is the radial component of a second force "A" below, applied transversely at each external thread 14 of each flange 12a, where each distance between an external thread on the
10 flange 12a where the first force "R" is applied and the base of the flange is designated by the length "l"; and (2) a second force "A" applied longitudinally along each flange 12a to each of the external threads 14 on the flange 12a. In response to the first and second force, a first moment "M1" and a second moment "M2" is created for each force, and each moment "M" is equal to the force times a lever arm.

15 The first moment "M1", associated with the first force "R", represents a torque applied to the base of each flange and is equal to the first force "R" times a first lever arm, the first lever arm being the length "l", which is a length between the application of force "R" at each of the external threads 14 and the base of the flange 12a. This first moment "M1" is desired and is not a problem.

20 However, the second moment "M2" associated with the second force "A" is equal to the second force "A" times the a second lever arm "delta" or " δ ". This second moment "M2" is a problem since it represents an undesired auxiliary bending moment or torque "M2" at the base of the flange which is created by a longitudinally directed second force "A" at each of the external threads 14. This undesired auxiliary bending moment "M2"
25 must be eliminated.

In figure 10, a view of a flange of the prior art snap latch of figure 5 is illustrated. In this figure, each of the flanges of the prior art snap latch of figure 5 is approximately rectangular in cross section and it has a width of "t". The desired bending moment M1 is equal to the transversely applied first force "R" times the first lever arm "l", which is the
30 length between the point of application of the first force "R" at each of the external threads 14 and the base of the flange 12a. However, the undesired auxiliary bending moment "M2" is created by the longitudinally applied second force "A" times a second lever arm "delta-old". The second lever arm "delta-old" is the transverse distance between the center of gravity 30 of the flange 12a and the direction of the second force A.
35 Since the second lever arm "delta-old" is not zero, the undesired auxiliary bending moment "M2" is not zero. In order to eliminate the undesired auxiliary bending moment M2, each of the flanges of the prior art snap latch of figures 5 and 10 must be redesigned in a way which will force the lever arm "delta-old" to be approximately equal to zero.

In figure 11, a view of one of the flanges 12a of the snap latch 12 of figure 6 of the snap latch seal locator 10 of figure 1 in accordance with the present invention is illustrated. In
40

5 this figure, each of the flanges 12a of the snap latch 12 of figure 6 of the present invention is designed to make the base 12a1 thicker than the base of the prior art flange shown in figure 10. For purposes of illustration of this concept, assume that the base of the prior art flange shown in figure 10 has a width "t"; however, further assume that the base 12a1 of the flange 12a of the snap latch 12 of the present invention shown in figure 10 11 has a width "2t", which is twice the thickness of the base "t" in figure 10. However, the tip (not the base) of the flange of figure 10 has a width "t" and the tip of the flange of figure 11 also has a width "t". As a result, in figure 11, since the width of the base is "2t", but the width of the tip is "t", the center of gravity 32 of the flange 12a of the present invention has shifted upwardly in figure 11, relative to the location of the center of gravity 30 shown in the prior art flange of figure 10. This shift of the center of gravity 32 upwardly in figure 11 has decreased the length of the second lever arm "delta-new". Recall that the second lever arm associated with the prior art flange in figure 10 is "delta-old". However, the second lever arm associated with the flange of the present invention in figure 11 is now "delta-new". Due to the shift of the center of gravity 32 upwardly in 20 figure 11, "delta-new" is very much less than "delta-old". In fact, the second lever arm "delta-new" in figure 11 is approximately equal to zero. Therefore, since the second lever arm "delta-new" is approximately equal to zero, the undesired auxiliary bending moment "M2" in figure 11 is also approximately equal to zero.

Referring to figure 12, this figure illustrates an external thread on a flange 12a of the snap 25 latch 12 of figure 6 in contact with an internal thread 24 on an internal periphery of a mandrel 23 of the packer assembly 22.

A sixth improvement to the snap latch seal locator 10 of figure 1 of the present invention is represented by the following basic principle in accordance with one aspect of the present invention: the outer diameter (OD) of the flange 12a of the snap latch 12 is greater 30 than the inner diameter (ID) of the mandrel 23 on the packer assembly plus a tolerance stackup.

In figure 12, a flange 12a of the snap latch 12 of figure 6 has an outer diameter (OD) 34 and the mandrel 23 of the packer assembly 22 has an inner diameter (ID) 36. An external thread 14a and 14b on the flange 12a connects to the OD 34 of the flange 12a, and an 35 internal thread 24a and 24b on the mandrel 23 connects to the ID 36 of the mandrel 23. The external thread 14a, 14b on flange 12a mates with the internal thread 24a and 24b on mandrel 23.

In figure 12, this figure illustrates the OD 34 of flange 12a in direct contact with the ID 36 of mandrel 23, that is, OD 34 = ID 36. However, in reality, this is not the case. In 40 reality, the OD 34 is not equal to the ID 36. Due to the machining of the OD 34 of the

5 flange 12a, there exists a "tolerance stackup" 38 on the surface of the flange 12a. As a result of this tolerance stackup 38, the OD 34 is not equal to the ID 36; rather, an OD 34a is less than (or greater than) the ID 36 by an amount equal to the tolerance stackup 38. Therefore, in reality, the external thread 14a, 14b on flange 12a does not mate directly with the internal thread 24a, 24b on mandrel 23. This increases the range of loads
10 required for engaging the tubing string to the packer assembly and for disengaging the tubing string from the packer assembly. This is a problem which must be solved.

Therefore, in accordance with the sixth improvement to the snap latch seal locator 10 of the present invention, in order to take into account the existence of this "tolerance stackup", the following basic principle must be followed when manufacturing the flanges
15 12a of the snap latch 12 of the snap latch seal locator 10 and the mandrel 23 of the packer assembly 22:

$$\text{OD } 34 > \text{ID } 36 + \text{tolerance stackup } 38$$

The above basic principle indicates that the outer diameter 34 of the flange 12a in figure 12 should be greater than the inner diameter 36 of the mandrel 23 plus the
20 aforementioned tolerance stackup 38. If this basic principle is adhered to when manufacturing the flanges 12a of the snap latch 12 and the mandrel 23 of the packer assembly 22, the external thread 14a, 14b on flange 12a will mate directly with the internal thread 24a, 24b on the mandrel 23 as required.

Referring to figures 13 and 14, the external threads on a flange of a prior art snap is illustrated in figure 13 and the external threads 14 on a flange 12a of the snap latch 12 of figure 6 of the snap latch seal locator 10 of figure 1 of the present invention is illustrated in figure 14.

In figure 13, a prior art flange 40 of the prior art snap latch of figure 5 includes a multitude of external threads 42. Each of the external threads 42 are disposed a distance
30 "I" from the base of the flange 40; for example, a first one of the external threads 42 lies a distance "I1" from the base 40a of the flange 40, a second one of the external threads 42 lies a distance "I2" from the base 40a, ..., and an nth one of the external threads 42 lies a distance "In" from the base 40a. Since each of the external threads 42 lies a different distance from the base 40a, each external thread 42 is associated with a different moment "M" at the base 40a since each external thread 42 has a force "R" applied
35 transversely thereto and each external thread 42 has a different lever arm "I". For example, the first one of the external threads has a force "R1" applied transversely thereto and lies at a lever arm distance of "I1" thereby developing a moment "M1" at the base 40a, and the second one of the external threads has a force "R2" applied transversely

5 thereto and lies at a lever arm distance of "l2" thereby developing a moment "M2" at the base 40a, etc. If there are "n" external threads 42, there are "n" moments (M1, M2, ..., Mn) developed at the base 40a of the flange 40.

The problem with the snap latch of the prior art shown in figures 5 and 13 is as follows: there are too many moments Mn developed at the base 40a of the flange 40; and, as a
10 result, too many disengaging loads are required to disengage the snap latch of the prior art shown in figure 5 from the packer assembly 22 of figure 2. Since there are too many moments Mn developed at the base 40a of the flange 40, it is too hard to predict what load will engage the tubing to the packer assembly and/or disengage the tubing from the packer assembly.

15 A seventh improvement to the snap latch seal locator 10 of figure 1 relates to the elimination of some of the external threads 42 from the prior art flange 40 of figure 13 thereby decreasing the number of moments developed at the base 40a of the flange 40 and thereby further decreasing the number of disengaging loads which are required to disengage the snap latch 12 of the snap latch seal locator 10 of figure 1 from the packer
20 assembly 22 of figure 2.

Accordingly, referring to figures 13 and 14, the external threads 42a on the prior art flange of the prior art snap latch of figure 13, which lie within the interval 42b, have been eliminated. As a result, the external threads 14 of the snap latch 12 of the present invention of figure 14 are shorter in length than the external threads 42 of the prior art
25 snap latch of figure 13 by an amount equal to the external threads 42a which lie within the interval 42b. As a result, the number of disengaging loads, required to disengage the snap latch 12 of the present invention of figure 6 from the packer assembly 22, has been minimized; and, since the number of disengaging loads has been minimized, the pull out force, required to pull the tubing 20 out of the packer assembly 22 of figures 1 and 2, can
30 be maintained closer to 15000 pounds.

The invention being thus described, it will be obvious that the same may be varied in many ways. Such variations are not to be regarded as a departure from the spirit and scope of the invention, and all such modifications as would be obvious to one skilled in the art are intended to be included within the scope of the following claims.

5

CLAIMS

1. A latch adapted for interconnecting a first part to a second part in a wellbore, comprising :
a base ;
10 a plurality of flanges connected to the base, the plurality of flanges defining a plurality of gaps disposed, respectively, between the plurality of flanges; and
an elastomeric barrier having a top part and a plurality of extension members connected to the top part, the plurality of extension members being disposed, respectively, within the plurality of gaps between the plurality of flanges.
- 15 2. The latch of claim 1, further comprising :
a plurality of external threads disposed around an external periphery of said plurality of flanges, each of said external threads having an engaging surface and a disengaging surface, an angle between said disengaging surface and a vertical lying in a range between approximately twenty (20) degrees and approximately twenty
20 eight (28) degrees.
3. The latch of claim 1 or claim 2, further comprising :
a snap latch body disposed adjacent to said plurality of flanges, a space being defined by said flanges and said snap latch body, a distance between said flanges and said snap latch body being a distance "t", said flanges being adapted to bend a
25 distance "x" across said space, said distance "x" being less than said distance "t".
4. The latch of claim 3, wherein said distance "t" lies in a range between 2-1/2 times said distance "x" to 3 times said distance "x".
5. The latch of claim 1, further comprising :
a plurality of external threads disposed around an external periphery of said
30 plurality of flanges, an axial force being adapted to be applied axially along a surface of said flanges to said plurality of external threads, said flanges each having a center of gravity, a transverse distance between said center of gravity and said axial force being approximately equal to zero.
6. A latch adapted for interconnecting a first part to a second part in a wellbore, comprising :
35 a base ;

5 a plurality of flanges connected to the base, the plurality of flanges defining a plurality of gaps disposed, respectively, between the plurality of flanges; and
a plurality of external threads disposed around an external periphery of said plurality of flanges, each of said external threads having an engaging surface and a disengaging surface, an angle between said disengaging surface and a vertical lying
10 in a range between approximately twenty (20) degrees and approximately twenty eight (28) degrees.

7. A latch adapted for interconnecting a first part to a second part in a wellbore, comprising :

a base ;

15 a plurality of flanges connected to the base, the plurality of flanges defining a plurality of gaps disposed, respectively, between the plurality of flanges; and
a snap latch body disposed adjacent to said plurality of flanges, a space being defined by said flanges and said snap latch body, a distance between said flanges and said snap latch body being a distance "t", said flanges being adapted to bend a distance "x" across said space, said distance "x" being less than said distance "t".
20

8. The latch of claim 7, wherein said distance "t" lies in a range between 2-1/2 times said distance "x" to 3 times said distance "x".

9. A latch adapted for interconnecting a first part to a second part in a wellbore, comprising :

25 a base ;

a plurality of flanges connected to the base, the plurality of flanges defining a plurality of gaps disposed, respectively, between the plurality of flanges; and
a plurality of external threads disposed around an external periphery of said plurality of flanges, an axial force being adapted to be applied axially along a surface of said flanges to said plurality of external threads, said flanges each having
30 a center of gravity, a transverse distance between said center of gravity and said axial force being approximately equal to zero.

10. In a snap latch adapted to interconnect a tubing string to a packer in a wellbore, the packer including a mandrel having internal threads, the snap latch including a plurality of flanges having external threads adapted to mate with the internal threads
35 of said mandrel of said packer when said tubing string is interconnected to said

- 5 packer in said wellbore, said external threads including a second number of said external threads, the improvement comprising :
- a first number of said external threads on said flanges of said snap latch, said first number of said external threads being less than said second number of said external threads;
- 10 said first number of said external threads being adapted to mate with internal threads of said mandrel.
11. The snap latch of claim 10, wherein the outer diameter of said flanges is greater than the inner diameter of said mandrel plus a tolerance stackup, said tolerance stackup representing an error during a machining of said flanges of said snap latch.
- 15 12. A snap latch adapted to interconnect a tubing string to a wellbore apparatus in a wellbore, the wellbore apparatus including a mandrel having internal threads, the latch comprising:
- a plurality of flanges having external threads adapted to mate with the internal threads of said mandrel when said tubing string is interconnected to said wellbore apparatus in said wellbore,
- 20 the outer diameter of each of said flanges being greater than the inner diameter of said mandrel plus a tolerance stackup, said tolerance stackup representing an error differential resulting from a machining of said flanges of said snap latch.

$$\frac{1}{2} \left(\frac{1}{2} \right)^{n-1} = \frac{1}{2^n} \quad \text{for } n \geq 1 \quad \text{and} \quad \frac{1}{2} \left(\frac{1}{2} \right)^0 = \frac{1}{2} \quad \text{for } n = 0$$



Application No: GB 9603857.5
Claims searched: 1-5

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Patents Act 1977
Search Report under Section 17

Databases searched:

UK Patent Office collections, including GB, EP, WO & US patent specifications, in:
UK CI (Ed.O): F2G (G4J, G4K, G4Z)
Int CI (Ed.6): E21B 17/02, 17/04, 17/046, 17/08, 33/038; F16L 37/00, 37/08,
37/084, 37/12, 37/133
Other:

Documents considered to be relevant:

Category	Identity of document and relevant passage	Relevant to claims
	None	

X	Document indicating lack of novelty or inventive step	A	Document indicating technological background and/or state of the art.
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